Simple Harmonic Motion

Physics Enhancement Programme for Gifted Students

The Hong Kong Academy for Gifted Education and Department of Physics, HKBU

Simple harmonic motion

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In mechanical physics, **simple harmonic motion** is a type of *periodic motion* where the restoring force is directly proportional to the displacement. *No matter what the direction of the displacement, the force always acts in a direction to restore the system to its equilibrium position.* It can serve as a mathematical model of a variety of motions, such as the oscillation of a spring, motion of a simple pendulum as well as molecular vibration.

 \mathfrak{p}

Mathematics of simple harmonic motion

Simple harmonic motion is a type of *periodic motion* which can use mathematical model to express it.

Time (t)

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Mathematics of simple harmonic motion

Since the motion returns to its initial value after one period T,

Displacement

FIG. 15-3 In all three cases, the blue curve is obtained from Eq. 15-3 with $\phi = 0$. (a) The red curve differs from the blue curve *only* in that the redcurve amplitude x'_m is greater (the red-curve extremes of displacement are higher and lower). (b) The red curve differs from the blue curve only in that the red-curve period is $T' = T/2$ (the red curve is compressed horizontally). (c) The red curve differs from the blue curve *only* in that for the red curve $\phi = -\pi/4$ rad rather than zero (the negative value of ϕ shifts the red curve to the right).

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 (c)

 $\overline{\pi}$

Mathematics of simple harmonic motion

Velocity $(t) = \frac{dx}{t} = \frac{a}{t} [x_m \cos(\omega t + \phi)]$ *dt d dt dx* $v(t) = \frac{dx}{dt} = \frac{dt}{dt} \left[x_m \right]$ $v(t) = -\omega x_m \sin(\omega t + \phi)$ Velocity amplitude $v_m = \omega x_m$ *Acceleration* Acceleration amplitude $(t) = \frac{dv}{dt} = \frac{dv}{dt}[-\omega x_m \sin(\omega t + \phi)]$ *dt d dt dv* $a(t) = \frac{dv}{dt} = \frac{u}{L}[-\omega x_m]$ $a(t) = -\omega^2 x_m \cos(\omega t + \phi) = -\omega^2 x(t)$ $a_m = \omega^2 x_m$

Equation of motion

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$$
\frac{d^2x}{dt^2} + \omega^2 x = 0
$$

This equation of motion will be very useful in identifying simple harmonic motion and its frequency.

Displacement $+x_m$ Ω $-x_m$ (a) $+\omega x_m$ Velocity $-\omega x_m$ (b) $\begin{bmatrix}\n\frac{5}{2} & +\omega^2 x_m \\
\frac{5}{2} & 0 \\
\frac{6}{2} & -\omega^2 x_m\n\end{bmatrix}$ (c)

> **FIG. 15-4** (a) The displacement $x(t)$ of a particle oscillating in SHM with phase angle ϕ equal to zero. The period T marks one complete oscillation. (b) The velocity $v(t)$ of the particle. (c) The acceleration $a(t)$ of the particle.

Simple harmonic motion of spring

Simple harmonic motion of a mass on a spring is subject to the linear elastic restoring force given by Hooke's Law. The motion is sinusoidal in time and demonstrates a single resonant frequency.

For one-dimensional simple harmonic motion, the equation of motion, which is a second-order linear ordinary differential equation with constant coefficients, could be obtained by means of Newton's second law and Hooke's law.

where m is the inertial mass of the oscillating body, x is its displacement from the equilibrium, and k is the spring constant.

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Simple harmonic motion of spring

Comparing with the equation of motion for simple harmonic motion,

Simple harmonic motion is the motion executed by a particle of mass m subject to a force that is proportional to the displacement of the particle but opposite in sign.

- A block whose mass *m* is 680 g is fastened to a spring whose spring constant k is 65 Nm⁻¹. The block is pulled a distance $x = 11$ cm from its equilibrium position at *x* = 0 on a frictionless surface and released from rest at *t* = 0.
- What are the angular frequency, the frequency, and the period of the resulting oscillation?
- What is the amplitude of the oscillation?
- What is the maximum speed of the oscillating block?
- What is the magnitude of the maximum acceleration of the block?
- What is the phase constant ϕ for the motion?
- *What is the displacement function x*(t)?

(a)

$$
\omega = \sqrt{\frac{k}{m}} = \sqrt{\frac{65}{0.68}} = 9.78 \text{ rad/s}^{-1} \qquad f = \frac{\omega}{2\pi} = 1.56 \text{ Hz} \qquad T = \frac{1}{f} = 0.643 \text{ s}
$$

 $x_m = 11$ cm (b)

(c)
$$
v_m = \omega x_m = (9.78)(0.11) = 1.08 \text{ ms}^{-1}
$$

(d)
$$
a_m = \omega^2 x_m = (9.78)^2 (0.11) = 10.5 \text{ ms}^{-2}
$$

(e) At $t = 0$, $x(0) = x_m \cos \phi = 0.11$ $v(0) = -\omega x_m \sin \phi = 0$ $\sin \phi = 0 \implies \phi = 0$

(f)
$$
x(t) = x_m \cos(\omega t + \phi) = 0.11 \cos(9.78t)
$$

At *t =* 0, the displacement of *x*(0) of the block in a linear oscillator is -8.50 cm. Its velocity $v(0)$ then is -0.920 ms⁻¹, and its acceleration $a(0)$ is +47.0 ms⁻².

What are the angular frequency ω ?

What is the phase constant ϕ *and amplitude* x_m *?*

$$
x(t) = xm \cos(\omega t + \phi) \qquad v(t) = -\omega xm \sin(\omega t + \phi) \qquad a(t) = -\omega^2 xm \cos(\omega t + \phi)
$$

At $t = 0$, $x(0) = x_m \cos \phi = -0.085$ (1) $v(0) = -\omega x_m \sin \phi = -0.920$ $a(0) = -\omega^2 x_m \cos \phi = +47.0$ 2 (0) (0) $=-\omega$ *x* $\frac{a(0)}{a^2} = -\omega^2$ and $a = \sqrt{a(0)} = \sqrt{47.0} = -23.5 \text{ rad}^{-1}$ 23.5 rads 0.0850 47.0 (0) (0) | 47.0 -23.5 rad^{-1} $=$ $\overline{}$ $=\sqrt{-\frac{u(v)}{2}}=\sqrt{-\frac{u(v)}{2}}$ *x* $\omega = \begin{vmatrix} a & b \\ -a & c \end{vmatrix}$ (2) (3) $(3) \div (1)$:

(2) ÷ (1):
\n
$$
\frac{v(0)}{x(0)} = -\omega \frac{\sin \phi}{\cos \phi} = -\omega \tan \phi
$$
\n
$$
\tan \phi = -\frac{v(0)}{\omega x(0)} = -\frac{-0.920}{(23.51)(-0.085)} = -0.4603
$$
\n
$$
\phi = -24.7^{\circ} \qquad \text{or} \qquad \phi = 180^{\circ} - 24.7^{\circ} = 155^{\circ}
$$
\n(1):
\n
$$
x_m = \frac{x(0)}{\cos \phi} \qquad \text{If } \phi = -24.7^{\circ}, \qquad x_m = \frac{-0.085}{\cos 24.7^{\circ}} = -0.094 \text{ m} = -9.4 \text{ cm}
$$
\n
$$
\text{If } \phi = 155^{\circ}, \qquad x_m = \frac{-0.085}{\cos 155^{\circ}} = 0.094 \text{ m} = 9.4 \text{ cm}
$$

Since x_m is positive, $\phi = 155^\circ$ and $x_m = 9.4$ cm.

Energy in Simple Harmonic Motion

Potential energy

$$
\therefore \qquad x(t) = x_m \cos(\omega t + \phi)
$$

$$
U(t) = \frac{1}{2}kx^2 = \frac{1}{2}kx_m^2 \cos^2(\omega t + \phi)
$$

Kinetic energy

$$
\therefore \qquad v(t) = -\omega x_m \sin(\omega t + \phi)
$$

$$
K(t) = \frac{1}{2}mv^2 = \frac{1}{2}m\omega^2 x_m^2 \sin^2(\omega t + \phi)
$$

$$
\therefore \qquad \omega = k/m
$$

$$
K(t) = \frac{1}{2}kx_m^2 \sin^2(\omega t + \phi).
$$

Energy in Simple Harmonic Motion

Mechanical energy

$$
E = U + K = \frac{1}{2} k x_m^2 \cos^2(\omega t + \phi) + \frac{1}{2} k x_m^2 \sin^2(\omega t + \phi)
$$

= $\frac{1}{2} k x_m^2 [\cos^2(\omega t + \phi) + \sin^2(\omega t + \phi)]$

$$
\therefore \qquad [\cos^2(\omega t + \phi) + \sin^2(\omega t + \phi)] = 1
$$

$$
E = U + K = \frac{1}{2}kx_m^2
$$

The mechanical energy is conserved !

FIG. 15-6 (a) Potential energy $U(t)$, kinetic energy $K(t)$, and mechanical energy E as functions of time t for a linear harmonic oscillator. Note that all energies are positive and that the potential energy and the kinetic energy peak twice during every period. (b) Potential energy $U(x)$, kinetic energy $K(x)$, and mechanical energy E as functions of position x for a linear harmonic oscillator with amplitude x_m . For $x = 0$ the energy is all kinetic, and for $x = \pm x_m$ it is all potential.

Suppose the damper of a tall building has mass $m = 2.72 \times 105$ kg and is designed to oscillate at frequency $f = 10$ Hz and with amplitude $x_m =$ 20 cm.

(a) What is the total mechanical energy *E* of the damper?

(b) What is the speed of the damper when it passes through the equilibrium point?

(a)
$$
k = m\omega^2 = m(2\pi f)^2
$$

= $(2.72 \times 10^5)(20\pi)^2 = 1.073 \times 10^9$ N

The energy:

$$
E = K + U = \frac{1}{2}mv^2 + \frac{1}{2}kx^2
$$

$$
= 0 + \frac{1}{2}(1.073 \times 10^9)(0.2)^2
$$

$$
= 2.147 \times 10^7 \text{ J} \approx 21.5 \text{ MJ}
$$

(b) Using the conservation of energy,

$$
E = K + U = \frac{1}{2}mv^{2} + \frac{1}{2}kx^{2}
$$

2.147 × 10⁷ = $\frac{1}{2}$ (2.72 × 10⁵) v^{2} + 0
 $v = 12.6$ ms⁻¹

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An Angular Simple Harmonic Oscillator

When the suspension wire is twisted through an angle θ , the torsional pendulum produces a restoring torque given by

 κ is called the torsion constant. Using Newton's law for angular motion,

$$
\tau = I\alpha,
$$

$$
-\kappa\theta = I\alpha,
$$

$$
\frac{d^2\theta}{dt^2} + \frac{\kappa}{I}\theta = 0.
$$

Comparing with the equation of motion for simple harmonic motion,

 $\tau = -\kappa\theta$. It is called the torsion constant.

Using Newton's law for angular motion,
 $\tau = I\alpha$,
 $-\kappa\theta = I\alpha$,
 $\frac{d^2\theta}{dt^2} + \frac{\kappa}{I}\theta = 0$.

Omparing with the equation of motion for
 $\omega^2 = \frac{\kappa}{I}$.
 $\omega^2 = \frac{\kappa}{I}$. $\omega^2 = \frac{\kappa}{\sqrt{2}}$ *I* . *T* 2π ω , *T I* $=2\pi$ K . Since

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A thin rod whose length *L* is 12.4 cm and whose mass *m* is 135 g is suspended at its midpoint from a long wire. Its period T_a of angular SHM is measured to be 2.53 s. An irregularly shaped object, which we call *X*, is then hung from the same wire, and its period T_b is found to be 4.76 s. What is the rotational inertia of object *X* about its suspension axis?

Rotational inertia of the rod about the center

$$
= I_a = \frac{1}{12}ML^2
$$

= $\left(\frac{1}{12}\right)(0.135)(0.124)^2$
= 1.7298×10⁻⁴ kgm²

K π ¹⁴ *a* $T_a = 2$ Since $T_a = 2\pi \sqrt{\frac{I_a}{I_a}}$ and $T_b = 2$

I

Thus
$$
\frac{T_a}{T_b} = \sqrt{\frac{I_a}{I_b}}
$$

Therefore,

Two torsion IG. 15-8 endulums, consisting of a) a wire and a rod and \dot{b}) the same wire and an tregularly shaped obect.

K

I

 π ¹

b

$$
\begin{aligned} \n\overset{b}{=} \left(T_a\right)^{-a} \\
&= \left(\frac{4.76}{2.53}\right)^2 (1.73 \times 10^{-4}) = 6.12 \times 10^{-4} \text{ kgm}^2 \n\end{aligned}
$$

a

2

 $\overline{}$

 \setminus

b $\mathbf{I}_b = \frac{\mathbf{I}_b}{\mathbf{T}} \mid I$

T

 $\overline{}$

 $\bigg($ $=$

The Simple Pendulum

The restoring torque about the point of suspension is $\tau = -mg \sin \theta L$. *Using Newton's law for angular motion,* $\tau = I\alpha$ *,*

 $-mg \sin \theta L = mL^2 \alpha$

$$
\frac{d^2\theta}{dt^2} + \frac{g}{L}\sin\theta = 0.
$$

When the pendulum swings through a small angle, $\sin \theta \approx \theta$. Therefore

 $\frac{\partial}{\partial z} + \frac{\delta}{I} \theta = 0.$ 2 $+\frac{8}{4} \theta =$ θ *L g dt d*

Comparing with the equation of motion for simple harmonic motion,

$$
\omega^2 = \frac{g}{L}.
$$

Since
$$
T = \frac{2\pi}{\omega}
$$
, $T = 2\pi \sqrt{\frac{L}{g}}$.

The Physical Pendulum

The restoring torque about the point of suspension is $\tau = -mg \sin \theta h$. *Using Newton's law for angular motion,* $\tau = I\alpha$,

 $-mg\sin\theta h = I\alpha$,

$$
\frac{d^2\theta}{dt^2} + \frac{mgh}{I}\sin\theta = 0.
$$

When the pendulum swings through a small angle, $\sin \theta \approx \theta$. Therefore

$$
\frac{d^2\theta}{dt^2} + \frac{mgh}{I}\theta = 0.
$$

Comparing with the equation of motion for simple harmonic motion,

FIG. 15-10 A physical pendulum. The restoring torque is $hF_g \sin \theta$. When $\theta = 0$, center of mass C hangs directly below pivot point O.

If the mass is concentrated at the center of mass *C*, such as in the simple pendulum, then

$$
T = 2\pi \sqrt{\frac{I}{mgh}} = 2\pi \sqrt{\frac{mL^2}{mgL}} = 2\pi \sqrt{\frac{L}{g}}.
$$

We recover the result for the simple pendulum.

A meter stick, suspended from one end, swings as a physical pendulum.

- (a) What is its period of oscillation *T*?
- (b) A simple pendulum oscillates with the same period as the stick. What is the length L_0 of the simple pendulum?

 (a) A meter stick suspended from one end as a FIG. 15-11 physical pendulum. (b) A simple pendulum whose length L_0 is chosen so that the periods of the two pendulums are equal. enter of oscillation. 22 (a) Rotational inertia of a rod about one end

$$
= \frac{1}{3}ML^{2}
$$

Period
$$
T = 2\pi \sqrt{\frac{I}{mgh}}
$$

$$
= 2\pi \sqrt{\frac{mL^{2}/3}{mgL/2}}
$$

$$
= 2\pi \sqrt{\frac{2L}{3g}}
$$

(b) For a simple pendulum of length L_0 ,

$$
T=2\pi\sqrt{\frac{L_0}{g}}
$$

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$$
2\pi \sqrt{\frac{L_0}{g}} = 2\pi \sqrt{\frac{2L}{3g}}
$$

\n
$$
\Rightarrow L_0 = \frac{2}{3}L = 66.7 \text{ cm}
$$

FIG. 15-11 (a) A meter stick suspended from one end as a physical pendulum. (b) A simple pendulum whose length L_0 is chosen so that the periods of the two pendulums are equal. Point P on the pendulum of (a) marks the center of oscillation.

A physical pendulum has a radius of gyration *k*. When it is suspended at distances *l* and *l'* from the center of mass, the periods of oscillation are the same. (a) Find the relation between *l* and *l'*. (b) This has been used to determine *g* accurately. Find an expression for *g*.

(a) When it is suspended at a distance *l* from the center of mass,

$$
I = Mk^{2} + Ml^{2}
$$
\n
$$
T = 2\pi \sqrt{\frac{Mk^{2} + Ml^{2}}{Mgl}} = 2\pi \sqrt{\frac{k^{2} + l^{2}}{gl}}
$$
\nSimilarly,
$$
T' = 2\pi \sqrt{\frac{k^{2} + l^{2}}{gl}}
$$
\nEquating *T* and *T'*,
$$
\frac{k^{2} + l^{2}}{l} = \frac{k^{2} + l^{2}}{l}
$$
\n
$$
l'k^{2} + l^{2}l' = lk^{2} + ll^{2}
$$
\n
$$
ll' = k^{2}
$$

(b) Substituting into the expression of *T*,

$$
T = 2\pi \sqrt{\frac{k^2 + l^2}{gl}} = 2\pi \sqrt{\frac{l + l^2}{g}}
$$

$$
g = 4\pi^2 \frac{l + l^2}{T^2}
$$

A diver steps on the diving board and makes it move downwards. As the board rebounds back through the horizontal, she leaps upward and lands on the free end just as the board has completed 2.5 oscillations during the leap. (With such timing, the diver lands when the free end is moving downward with greatest speed. The landing then drives the free end down substantially, and the rebound catapults the diver high into the air.) Modeling the spring board as the rodspring system (Fig. 15-12(d)), what is the required spring constant *k*? Given *m* = 20 kg, diver's leaping time t_f = 0.62 s.

 (a) A diving board. (b) The diver leaps upward FIG. 15-12 and forward as the board moves through the horizontal. (c) The diver lands 2.5 oscillations later. (d) A spring-oscillator model of the oscillating board. Hong Kong Baptist University

When the board is displaced by an angle θ ,

The restoring torque:

$$
\tau = -kxL = -kL^2 \sin \theta
$$

$$
\approx -kL^2 \theta
$$

Using Newton's law for angular motion,

$$
\tau = I\alpha
$$

$$
-kL^2\theta = \frac{1}{3}mL^2\frac{d^2\theta}{dt^2}
$$

$$
\Rightarrow \frac{d^2\theta}{dt^2} + \frac{3k}{m}\theta = 0
$$

Comparing with the equation of motion for simple harmonic motion,

2.5

0.62

$$
\omega^2 = \frac{3k}{m} \qquad \Rightarrow \qquad k = \frac{m\omega^2}{3} = \frac{m}{3} \left(\frac{2\pi}{T}\right)^2
$$

2.5 $=\frac{r_{fl}}{1-r}$ t_{fl} *T* The period should be

$$
\tau = -kxL = -kL^2 \sin \theta
$$

\n
$$
\approx -kL^2 \theta
$$

\nsing Newton's law for angular motion,
\n
$$
\tau = I\alpha
$$

\n
$$
-kL^2 \theta = \frac{1}{3}mL^2 \frac{d^2 \theta}{dt^2}
$$

\n
$$
\Rightarrow \frac{d^2 \theta}{dt^2} + \frac{3k}{m} \theta = 0
$$

\nomparing with the equation of motion for simple harmonic motion,
\n
$$
\omega^2 = \frac{3k}{m} \qquad \Rightarrow \qquad k = \frac{m\omega^2}{3} = \frac{m}{3} \left(\frac{2\pi}{T}\right)^2
$$

\nThe period should be
$$
T = \frac{t_{\beta}}{2.5} = \frac{0.62}{2.5}
$$

\nTherefore
$$
k = \frac{20}{2.5} \left(\frac{2\pi}{0.62/2.5}\right)^2 = 4280 \text{ Nm}^{-1}
$$

Damped Simple Harmonic Motion

The liquid exerts a damping force proportional to the velocity. Then,

 $F_d = -bv$, $b =$ damping constant.

Using Newton's second law,

$$
-bv - kx = ma.
$$

$$
m\frac{d^2x}{dt^2} + b\frac{dx}{dt} + kx = 0.
$$

Solution:
$$
x(t) = x_m e^{-bt/2m} \cos(\omega' t + \phi)
$$
,

where
$$
\omega = \sqrt{\frac{k}{m} - \frac{b^2}{4m^2}}
$$
.

If $b = 0$, ω reduces to $\omega = \sqrt{k/m}$

of the undamped oscillator.

If
$$
b \ll \sqrt{km}
$$
, then $\omega' \approx \omega$.

 $bt/2m$ The amplitude, $x(t) = x_m e^{-bt/2m}$, gradually decreases with time.

The mechanical energy decreases exponentially with time.

. 2 1 $E(t) = \frac{1}{2} kx_m^2 e^{-bt/m}$

15-8 | Damped Simple Harmonic Motion

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FIG. 15-16 The displacement function $x(t)$ for the damped oscillator of Fig. 15-15, with values given in Sample Problem 15-7. The amplitude, which is $x_m e^{-bt/2m}$, decreases exponentially with time.

For the damped oscillator with $m = 250$ g, $k = 85$ Nm⁻¹, and $b = 70$ gs⁻¹.

(a) What is the period of the motion?

(b) How long does it take for the amplitude of the damped oscillations to drop to half its initial value?

(c) How long foes it take for the mechanical energy to drop to half its initial value?

(a)
$$
T = 2\pi \sqrt{\frac{m}{k}} = 2\pi \sqrt{\frac{0.25}{85}} = 0.34 \text{ s}
$$

(b) When the amplitude drops by half, $x_m e^{-bt/2m} =$

itude drops by half,
$$
x_m e^{-bt/2m} = \frac{1}{2}x_m
$$

\n
$$
e^{-bt/2m} = \frac{1}{2}
$$
\nTaking logarithm, $-\frac{bt}{2m} = \ln \frac{1}{2} = -\ln 2$
\n
$$
t = \frac{2m \ln 2}{b} = \frac{(2)(0.25)(\ln 2)}{0.07} = 4.95 \text{ s}
$$

bt $/2m$

 $/2m=1$

(c) When the energy drops by half,

$$
\frac{1}{2}kx_m^2e^{-bt/m} = \frac{1}{2}\left(\frac{1}{2}kx_m^2\right)
$$

$$
e^{-bt/m} = \frac{1}{2}
$$

Taking logarithm,
$$
-\frac{bt}{m} = \ln \frac{1}{2} = -\ln 2
$$

$$
t = \frac{m \ln 2}{b} = \frac{(0.25)(\ln 2)}{0.07} = 2.48 \text{ s}
$$

Forced Oscillations and Resonance

When a simple harmonic oscillator is driven by a periodic external force, we have *forced oscillations* or *driven oscillations*.

Its behavior is determined by *two* angular frequencies:

(1) the *natural* angular frequency $\omega = \sqrt{k/m}$

The motion of the forced oscillator is given by $x(t) = x_m \cos(\omega_d t + \phi)$. (2) the angular frequency ω_d of the external driving force.

Substituting into the equation of motion,

$$
m\frac{d^2x}{dt^2} + b\frac{dx}{dt} + kx = F\cos\omega_d t,
$$

$$
x_m[(k - m\omega_a^2)\cos(\omega_a t + \phi) - b\omega_a\sin(\omega_a t + \phi)] = F\cos\omega_a t.
$$

Using the identity $cos(A+B) = cos A cos B - sin A sin B$

the natural angular frequency
$$
\omega = \sqrt{k/m}
$$

\nthe angular frequency ω_d of the external driving force.
\nmotion of the forced oscillator is given by $x(t) = x_m \cos(\omega_d t + \phi)$.
\nSubstituting into the equation of motion,
\n $m\frac{d^2x}{dt^2} + b\frac{dx}{dt} + kx = F \cos \omega_d t$,
\n $x_n[(k - m\omega_a^2)\cos(\omega_a t + \phi) - b\omega_a \sin(\omega_a t + \phi)] = F \cos \omega_a t$.
\nUsing the identity $\cos(A + B) = \cos A \cos B - \sin A \sin B$
\n $\sqrt{(k - m\omega_a^2)^2 + b^2\omega_a^2} x_n \cos(\omega_a t + \phi + \alpha) = F \cos \omega_a t$,
\nwhere $\cos \alpha = \frac{b\omega_a}{\sqrt{(k - m\omega_a^2)^2 + b^2\omega_a^2}}$.
\nHence $x_n = \frac{F}{\sqrt{m^2(\omega^2 - \omega_a^2)^2 + b^2\omega_a^2}}$, and $\phi = -\alpha$.
\nThe Hence $x_n = \frac{F}{\sqrt{m^2(\omega^2 - \omega_a^2)^2 + b^2\omega_a^2}}$, and $\phi = -\alpha$.

(2) Its amplitude x_m is greatest when $\omega_d = \omega$. (1) It oscillates at the angular frequency $\omega_{\sf d}$ of the external driving force.

This is called **resonance**. See Youtube "Tacoma Bridge Disaster".

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The displacement ampli-**FIG. 15-17** tude x_m of a forced oscillator varies as the angular frequency ω_d of the driving force is varied. The curves here correspond to three values of the damping constant b.

Two Coupled Oscillators and Normal Coordinates

x

k x 1 2 *k k* 0 *m m* (), ¹ ¹ ⁰ ² ¹ *m x kx k x x* (). ² ² ⁰ ¹ ² *m x kx k x x i t x Ae* Re ¹ *i t x Be* Re ² Using Newton's second law, Possible solution, and (*A* and *B* are complex.)

It is convenient to adopt the third trial solution. Then

$$
(k + k_0 - m\omega^2)A - k_0B = 0,
$$

\n
$$
(k + k_0 - m\omega^2)B - k_0A = 0.
$$

\nFor non-trivial solutions, we have
\n
$$
\frac{A}{B} = \frac{k_0}{k + k_0 - m\omega^2} = \frac{k + k_0 - m\omega^2}{k_0}.
$$

\nMore generally, we can use the matrix form:
\n
$$
\begin{pmatrix}\nk + k_0 - m\omega^2 & -k_0 \\
-k_0 & k + k_0 - m\omega^2\n\end{pmatrix}\begin{pmatrix}A \\ B\end{pmatrix} = \begin{pmatrix}0 \\ 0\end{pmatrix},
$$
\n
$$
\begin{pmatrix}\n\frac{k}{B} + k_0 - m\omega^2 & -k_0 \\
-k_0 & k + k_0 - m\omega^2\n\end{pmatrix}\begin{pmatrix}B \\ B\end{pmatrix} = \begin{pmatrix}0 \\ 0\end{pmatrix},
$$
\n
$$
\begin{pmatrix}\n\frac{k}{B} + k_0 - m\omega^2 & -k_0 \\
-\frac{k_0}{B} - k_0 & \frac{k_0}{B}\end{pmatrix}\begin{pmatrix}0 \\ \frac{k_0}{B} & \frac{k_0}{B} \\ \frac{k_0}{B} & \frac{k_0}{B}\end{pmatrix} = \begin{pmatrix}0 \\ 0 \end{pmatrix},
$$
\n
$$
\begin{pmatrix}\n\frac{k_0}{B} + k_0 - m\omega^2 & \frac{k_0}{B} \\
-\frac{k_0}{B} & \frac{k_0}{B}\end{pmatrix} = \begin{pmatrix}0 \\ 0 \end{pmatrix},
$$
\n
$$
\begin{pmatrix}\n\frac{k_0}{B} + k_0 - m\omega^2 & \frac{k_0}{B} \\ \frac{k_0}{B} & \frac{k_0}{B}\end{pmatrix} = \begin{pmatrix}0 \\ 0 \end{pmatrix},
$$

and for non-trivial solutions,

$$
\begin{vmatrix} k + k_{\scriptscriptstyle 0} - m\omega^2 & -k_{\scriptscriptstyle 0} \\ -k_{\scriptscriptstyle 0} & k + k_{\scriptscriptstyle 0} - m\omega^2 \end{vmatrix} = 0.
$$

 $(k + k_{0} - m\omega^{2})^{2} - k_{0}^{2} = 0,$ 0 Either way, we arrive at a secular equation, $(k + k_0 - m\omega^2)^2 - k_0^2 =$

$$
\omega^2 = \frac{k}{m}, \qquad \text{or} \qquad \omega^2 = \frac{k + 2k_0}{m}.
$$

$$
\text{If} \quad \omega = \omega_{\text{\tiny{I}}} \equiv \sqrt{\frac{k}{m}}, \qquad A = B,
$$

If
$$
\omega = \omega_2 \equiv \sqrt{\frac{k+2k_0}{m}}
$$
, $A = -B$.

Hence we obtain two solutions. In each solution, the two particles oscillate with the same frequency. They are called *normal modes*. Their frequencies are called *normal frequencies*. Any other solutions are combinations of the normal modes.

k Symmetric mode: $\omega_1 = \sqrt{\frac{k}{m}}$ and $x_1 = x_2 = A\cos(\omega t + \phi)$

Antiymmetric mode:
$$
\omega_i = \sqrt{\frac{k + 2k_o}{m}}
$$
 $x_i = A \cos(\omega t + \phi)$ and $x_2 = -A \cos(\omega t + \phi)$

\nIn general, the mode that has the highest symmetry will have the lowest frequency, while the antisymmetric mode has the highest frequency.

\nthe symmetric mode can be excited by pulling the two particles from their equilibrium positions by equal amounts in the same direction so that $x_i(0) = x_2(0) = A$ and $\dot{x}_i(0) = \dot{x}_2(0) = 0$

\nthe antisymmetric mode can be excited by pulling apart the two particles equally in opposite directions and then released, so that $\dot{x}_i(0) = -x_2(0) = A$ and $\dot{x}_i(0) = \dot{x}_2(0) = 0$

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In general, the mode that has the highest symmetry will have the lowest frequency, while the antisymmetric mode has the highest frequency.

The symmetric mode can be excited by pulling the two particles from their equilibrium positions by equal amounts in the same direction so that

$$
x_1(0) = x_2(0) = A
$$
 and $\dot{x}_1(0) = \dot{x}_2(0) = 0$

. equally in opposite directions and then released, so that The antisymmetric mode can be excited by pulling apart the two particles

$$
x_1(0) = -x_2(0) = A
$$
 and $\dot{x}_1(0) = \dot{x}_2(0) = 0$

Find the frequencies of small oscillations of a double pendulum. Tangential component of forces acting on the upper particle:

 $F_t = -mg \sin \theta_1 + T \sin(\theta_2 - \theta_1)$

For small oscillations, $T \approx mg$,

L x_1 $\sin \theta_1 \approx$ *L* $x_2 - x_1$ $\sin \theta_2 \approx \frac{x_2 - x_1}{r}$ *L* $x_2 - 2x_1$ 2 V_1 2 $\sin(\theta_2 - \theta_1)$ \overline{a} $(\theta, -\theta) \approx$

Using Newton's second law,

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$$
mL\ddot{\theta}_1 = -F_t \implies m\ddot{x}_1 \approx -\frac{mg}{L}x_1 + \frac{mg}{L}(x_2 - 2x_1)
$$

$$
mL\ddot{\theta}_2 = -mg\sin\theta_2 \implies m\ddot{x}_2 = -\frac{mg}{L}(x_2 - x_1)
$$

$$
\begin{pmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{pmatrix} = \begin{pmatrix} -3g/L & g/L \\ g/L & -g/L \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix}
$$

Let
$$
\begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = \text{Re} \begin{pmatrix} A \\ B \end{pmatrix} e^{-i\omega t}
$$

Then $\begin{pmatrix} 3g/L - \omega^2 & -g/L \end{pmatrix} \begin{pmatrix} x \end{pmatrix}$

Then
$$
\begin{pmatrix} 3g/L - \omega^2 & -g/L \\ -g/L & g/L - \omega^2 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = 0
$$

0 $/L$ $g/$ $3g/L-\omega^2$ $-g/k$ 2 2 $=$ $-g/L$ g/L $-\omega^2$ ω ω g/L *g* $/L$ For non-trivial solutions, $\begin{array}{cc} \left|3g/L\!-\!\omega^2 & -g/L \end{array}$

Symmetric mode: $\omega^2 = (2 - \sqrt{2})g/L$ and $B = \sqrt{2}A$

Antisymmetric mode:
$$
\omega^2 = (2 + \sqrt{2})g/L
$$
 and $B = -\sqrt{2}A$

Consider two pendula of length *L* and mass *m* coupled by a spring with force constant *k*. Find the normal frequencies, the normal modes and the general solution.

Symmetric mode:

$$
\omega_1^2 = \frac{g}{L} \qquad , A = B
$$

Antisymmetric mode:

$$
\omega_2^2 = \frac{g}{L} + \frac{2k}{m} \qquad , A = -B
$$

