

## Thermal Expansion (Area)

The thermodynamics

$$\alpha = \frac{\Delta L/L}{\Delta T}$$

$$\beta = \frac{\Delta A/A}{\Delta T}$$

$$\gamma = \frac{\Delta V/V}{\Delta T}$$

$$\begin{aligned}
 A + \Delta A &= (L + \alpha L)(W + \alpha W) \\
 &= (L + \alpha L \Delta T)(W + \alpha W \Delta T) \\
 &= LW + \alpha LW \Delta T + \alpha LW \Delta T + \alpha^2 LW \Delta T^2 - ① \\
 &= LW(1 + 2\alpha \Delta T + \alpha^2 \Delta T^2) \\
 &= LW(1 + \alpha \Delta T)^2 \\
 &= A(1 + \alpha \Delta T)^2
 \end{aligned}$$

$$① \Rightarrow \Delta A = 2\alpha A \Delta T$$

$$\frac{\Delta A}{A} \approx 2\alpha \Delta T$$

$$\beta = \frac{\Delta A/A}{\Delta T} \approx \frac{2\alpha \Delta T}{\Delta T} = 2\alpha$$

## Thermal Expansion (Volume)

$$V + \Delta V = (L + \alpha L)(W + \alpha W)(H + \alpha H)$$

$$= (L + \alpha L \Delta T)(W + \alpha W \Delta T)(H + \alpha H \Delta T)$$

$$= LWH (1 + \alpha \Delta T)^3$$

$$\frac{\Delta V}{V} = 3\alpha \Delta T + 3(\alpha \Delta T)^2 + (\alpha \Delta T)^3$$

Suppose  $(\alpha \Delta T)^2 \ll 1$  and  $(\alpha \Delta T)^3 \ll 1$ ,

$$\Rightarrow \frac{\Delta V}{V} \approx 3\alpha \Delta T$$

$$\gamma = \frac{\Delta V/V}{\Delta T} = \frac{3\alpha \Delta T}{\Delta T} = 3\alpha$$

## Exercise : Hydrostatic Pressure on Solar Pond

$$P_1 = \rho g h_1 = (1040)(9.8)(0.8)$$

$$= 8.15 \text{ kN/m}^2 \quad (\text{or } 8.15 \text{ kPa})$$

Remark:  $1 \text{ kN/m}^2 = 1 \text{ kPa}$

$$dP = \rho g dz$$

$$P_2 - P_1 = \int_0^z \rho g dz$$

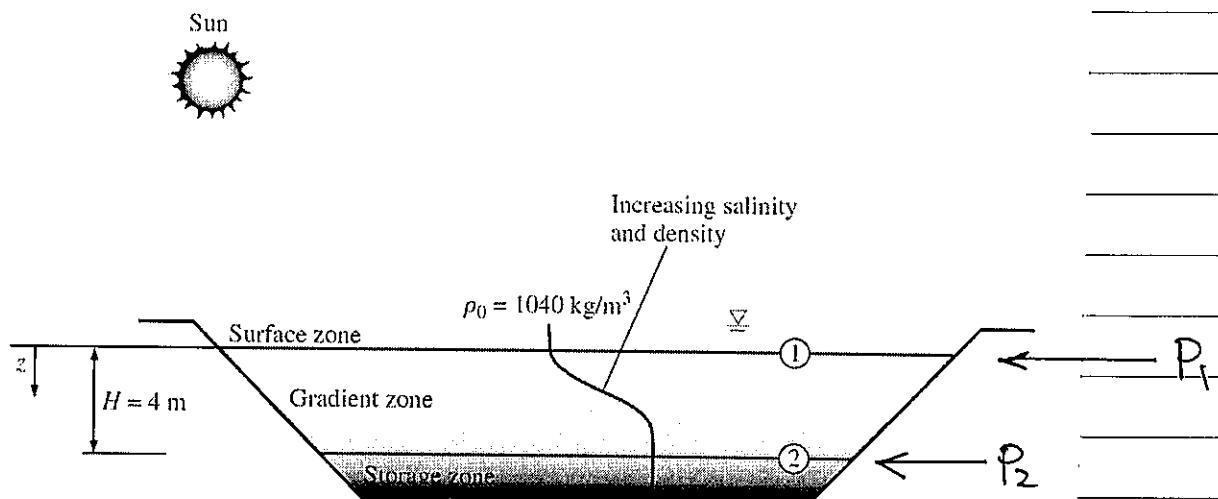
$$P_2 = P_1 + \int_0^z \rho_0 \left[ 1 + \tan^2 \left( \frac{\pi z}{4H} \right) \right] g dz$$

$$P_2 = P_1 + \left( \rho_0 g \frac{4H}{\pi} \right) \sinh^{-1} \left( \tan \frac{\pi z}{4H} \right)$$

$$= 8.15 \text{ kPa} + (1040)(9.8) \left( \frac{4 \times 4}{\pi} \right)$$

$$\sinh^{-1} \left( \tan \frac{\pi}{4} \cdot \frac{4}{4} \right)$$

$$= 53.9 \text{ kPa}$$

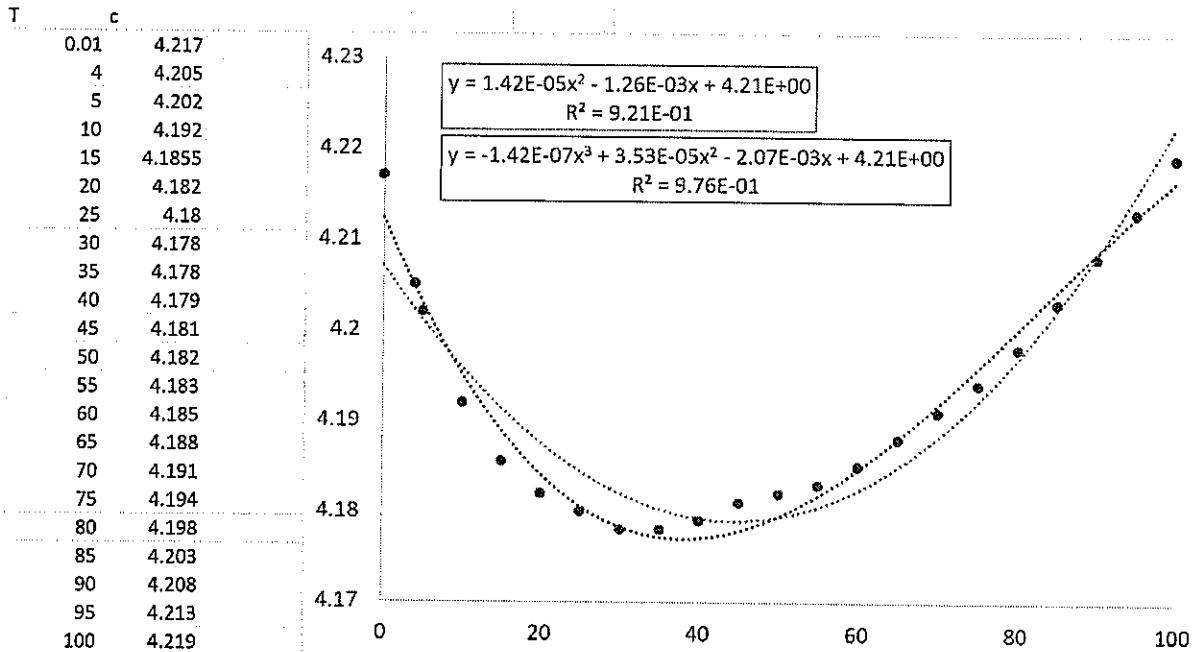


# Temperature Dependent Specific Heat

$$(a) \quad Q = (1)(4200)(99.99)$$

$$= \sim 420 \text{ kJ}$$

(b)



$$Q = m \int_{T_i}^{T_f} c(T) dT$$

$$\text{Fitting I : } Q = \int_{0.01}^{100} (1.42 \times 10^{-5} T^2 - 1.26 \times 10^{-3} T + 4.21) dT$$

$$= 419.391 \text{ kJ}$$

$$\text{Fitting II : } Q = \int_0^{100} (-1.42 \times 10^{-7} T^3 + 3.53 \times 10^{-5} T^2 - 2.07 \times 10^{-3} T + 4.21) dT$$

$$= 418.825 \text{ kJ}$$

## Isothermal Expansion

$$W = - \int_{V_i}^{V_f} P dV$$

$$= - \int_{V_i}^{V_f} \frac{nRT}{V} dV$$

$$= - nRT \int_{V_i}^{V_f} \frac{dV}{V}$$

$$= nRT \ln\left(\frac{V_f}{V_i}\right)$$

## Quick Quiz on Thermal Conductivity

In Series :  $R_{\text{eff, series}} = R_1 + R_2$

In Parallel : 
$$\begin{aligned} (R_{\text{eff, parallel}})^{-1} &= \frac{1}{2A} \left( \frac{A}{R_1} + \frac{A}{R_2} \right) \\ &= \frac{1}{2} \frac{R_1 + R_2}{R_1 R_2} \\ R_{\text{eff, parallel}} &= \frac{2 R_1 R_2}{R_1 + R_2} \end{aligned}$$

$$R_{\text{eff, series}} - R_{\text{eff, parallel}}$$

$$= \frac{2 R_1 R_2}{R_1 + R_2} - R_1 - R_2$$

$$= \frac{2 R_1 R_2}{R_1 + R_2} - (R_1 + R_2)(R_1 + R_2)$$

$$= \frac{2 R_1 R_2}{R_1 + R_2} - (R_1 + R_2)^2$$

$$= \frac{R_1^2 + R_2^2}{R_1 + R_2} > 0$$

$\Rightarrow$  Thermal Resistance  $R$  (in Series)  $>$   $R$  (in parallel)

## Exercise on Thermal Conductivity

Two slabs, thickness  $L_1$  and  $L_2$ , and thermal conductivities  $k_1$  and  $k_2$ , are in thermal contact with each other. The temperatures of their outer surfaces are  $T_c$  and  $T_h$ , respectively ( $T_h > T_c$ ). Determine the temperature,  $T$ , at the interface and the rate of energy transfer under steady-state condition.

Solution:

$$\frac{\Delta Q}{\Delta t} = k A \left( \frac{T_h - T_c}{L} \right)$$

$$\frac{\Delta Q_1}{\Delta t} = k_1 A \left( \frac{T - T_c}{L_1} \right) \quad \text{--- (1)}$$

$$\frac{\Delta Q_2}{\Delta t} = k_2 A \left( \frac{T_h - T}{L_2} \right) \quad \text{--- (2)}$$

At Steady State,  $\frac{\Delta Q_1}{\Delta t} = \frac{\Delta Q_2}{\Delta t}$

$$k_1 A \left( \frac{T - T_c}{L_1} \right) = k_2 A \left( \frac{T_h - T}{L_2} \right)$$

$$\Rightarrow T = \frac{k_1 L_2 T_c + k_2 L_1 T_h}{k_1 L_2 + k_2 L_1}$$

$$\frac{\Delta Q}{\Delta t} = A \left( \frac{T_h - T_c}{\left( \frac{L_1}{k_1} \right) + \left( \frac{L_2}{k_2} \right)} \right)$$

## Exercise on Thermal Conductivity II

The following hollow cylinder is filled with water and maintained at temperature  $T_a$ , whereas the outer surface is  $T_b$ . ( $T_a > T_b$ ) The wall of the cylinder has a thermal conductivity  $k$ . Find the rate of energy transfer from the inner wall to the outer surface.

Solution:

$$\frac{dQ}{dt} = -kA \frac{dT}{dx}$$

$$\frac{dQ}{dt} = -k (2\pi r L) \frac{dT}{dr}$$

$\frac{dQ}{dt}$  is constant,

$$\Rightarrow dT = \frac{dQ}{dt} \left( -\frac{1}{2\pi k r L} \right) dr$$

$$dT = \frac{dQ}{dt} \left( -\frac{1}{2\pi k L} \right) \frac{dr}{r}$$

$$\int_{T_a}^{T_b} dT = \frac{dQ}{dt} \left( -\frac{1}{2\pi k L} \right) \int_a^b \frac{dr}{r}$$

$$T_b - T_a = \frac{dQ}{dt} \left( -\frac{1}{2\pi k L} \right) \ln \left( \frac{b}{a} \right)$$

$$\Rightarrow \frac{dQ}{dt} = \frac{2(T_a - T_b) k \pi L}{\ln(b/a)}$$

## Exercise on Thermal Conductivity and Latent Heat

A pond of water at  $0^\circ\text{C}$  is covered with a layer of ice 4 cm thick. If the air temperature stays constant at  $-10^\circ\text{C}$ , how long does it take for the ice thickness to increase to 8 cm?

Solution:

Air at  $-10^\circ\text{C}$

$$L p A \frac{dx}{dt} = kA \left( \frac{\Delta T}{\Delta x} \right)$$

Ice

4cm

$$L p \int_4^8 x dx = k \Delta T \int dt$$

↓  
Another 4cm ice  
 $\Delta t = ?$

$$L p \left[ \frac{x^2}{2} \right]_4^8 = k \Delta T \Delta t$$

$$(334 \text{ J/kg}) (917) / \left( \frac{0.08^2 - 0.04^2}{2} \right) = (2)(10) \Delta t$$

$$\Rightarrow \Delta t = 36753 \text{ s} = 10.2 \text{ hours}$$

## Exercise on Molar Specific Heat

### Constant Volume

First Law :  $\Delta E_{\text{int}} = Q + \cancel{W}$

$$\Delta E_{\text{int}} = \frac{3}{2} nRT$$

Also :  $\Delta E_{\text{int}} = nC_V \Delta T$

$$C_V = \frac{1}{n} \frac{dE_{\text{int}}}{dT}$$

$$= \frac{1}{n} \left( \frac{3nR}{2} \right)$$

$$= \frac{3}{2} R$$

### Constant Pressure

$$\begin{cases} Q = nC_P \Delta T \\ W = -P \Delta V \end{cases}$$

$$\Delta E_{\text{int}} = Q + W$$

$$= nC_P \Delta T + (-P \Delta V)$$

$$= nC_P \Delta T - nR \Delta T$$

$$\Rightarrow nC_V \Delta T = nC_P \Delta T - nR \Delta T$$

$$\Rightarrow C_P - C_V = R$$

Exercise : Heating a Cylinder of Helium Gas

i)  $Q = n C_V \Delta T$

$$= (3 \text{ moles}) (12.5 \text{ J/mole} \cdot \text{K}) (200 \text{ K})$$

$$= 7.5 \text{ kJ}$$

ii)  $Q = n C_P \Delta T$

$$= (3 \text{ moles}) (20.8 \text{ J/mole} \cdot \text{K}) (200 \text{ K})$$

$$= 12.48 \text{ kJ}$$

Exercise : Adiabatic Process (to be continued)

$$PV = nRT$$

$$PdV + VdP = nR dT \quad \text{--- } ①$$

$$\Delta E_{int} = nC_V dT = -PdV \quad \text{--- } ②$$

Eliminating  $dT$  from Eqn. ① and ②,

$$PdV + VdP = nR \left( -\frac{PdV}{nC_V} \right)$$

$$PdV + VdP = -PR \left( \frac{dV}{C_V} \right)$$

$$PdV + VdP = -PR \frac{dV}{C_V}$$

$$VdP = \left( -P - \frac{PR}{C_V} \right) dV$$

$$\frac{dP}{P} = - \left( 1 + \frac{R}{C_V} \right) \frac{dV}{V}$$

$$\frac{dP}{P} = - \left( \frac{C_V + R}{C_V} \right) \frac{dV}{V}$$

$$\frac{dP}{P} = - \left( \frac{C_V + C_P - C_V}{C_V} \right) \frac{dV}{V}$$

$$\frac{dP}{P} = - \frac{C_P}{C_V} \frac{dV}{V}$$

$$\frac{dP}{P} + \gamma \frac{dV}{V} = 0$$

$$\Rightarrow \ln P + \gamma \ln V = \text{Constant}$$

$$\Rightarrow PV^\gamma = \text{Constant}$$

## Exercise : Adiabatic Process (continued)

$$\left\{ \begin{array}{l} P_i V_i^\gamma = P_f V_f^\gamma \\ P_i = \frac{n R T_i}{V_i} \quad \text{and} \quad P_f = \frac{n R T_f}{V_f} \end{array} \right.$$

$$\Rightarrow \frac{n R T_i}{V_i} (V_i^\gamma) = \frac{n R T_f}{V_f} (V_f^\gamma)$$

$$\Rightarrow T_i V_i^{\gamma-1} = T_f V_f^{\gamma-1}$$

## Exercise : Diesel Engine Cylinder

$$P_f = P_i \left( \frac{V_i}{V_f} \right)^{\gamma} = (1 \text{ atm}) \left( \frac{800}{60} \right)^{1.4}$$
$$= 37.6 \text{ atm}$$

$$\frac{P_i V_i}{T_i} = \frac{P_f V_f}{T_f}$$

$$T_f = \frac{P_f V_f}{P_i V_i} T_i = \frac{(37.6)(60)}{(1)(800)} (293)$$
$$= 826 \text{ K} \quad (553^\circ\text{C})$$

## Exercise on Number Density

$$\frac{n_v(E_2)}{n_v(E_1)} = \frac{N_0 \exp(-E_2/k_B T)}{N_0 \exp(-E_1/k_B T)}$$

$$= \exp\left(-\frac{E_2 - E_1}{k_B T}\right)$$

$$= \exp\left[-\frac{1.5 \text{ eV}}{(1.38 \times 10^{-23} \times 2500) / 1.6 \times 10^{-19}}\right]$$

$$= \exp(-6.96)$$

$$= 9.49 \times 10^{-4}$$

## Exercise on Energy Quality = The Entropy

1 kg water at 20°C is mixed with 1 kg water at 80°C.

Calculate the net change in Entropy.

Solution:

$$\begin{aligned}\text{Entropy Change } \Delta S &= \int_i^f \frac{dQ}{T} \\ &= \int_{T_i}^{T_f} \frac{MC dT}{T} \\ &= MC \ln\left(\frac{T_f}{T_i}\right)\end{aligned}$$

Final Temperature of the water mixture = 50°C

$$\Delta S_{\text{net}} = \Delta S_{20^\circ\text{C} \rightarrow 50^\circ\text{C}} + \Delta S_{80^\circ\text{C} \rightarrow 50^\circ\text{C}}$$

$$= MC \ln\left(\frac{323}{293}\right) + MC \ln\left(\frac{323}{353}\right)$$

$$= 36.54 \text{ J/K}$$

Exercise: Efficiency of the Carnot Engine

Process A  $\rightarrow$  B, Isothermal Expansion.

Energy transfer by heat from the hot reservoir.

$$|Q_h| = |\Delta E_{int} - W_{AB}| = |0 - W_{AB}| = W_{AB} = n R T_h \ln\left(\frac{V_B}{V_A}\right) \quad (1)$$

Process C  $\rightarrow$  D, Isothermal Compression.

Energy transfer to the cold reservoir.

$$|Q_c| = |\Delta E_{int} - W_{CD}| = |0 - W_{CD}| = W_{CD} = n R T_c \ln\left(\frac{V_D}{V_B}\right) \quad (2)$$

$$\frac{(2)}{(1)} \Rightarrow \frac{|Q_c|}{|Q_h|} = \frac{T_c}{T_h} \left[ \frac{\ln\left(\frac{V_c}{V_D}\right)}{\ln\left(\frac{V_B}{V_A}\right)} \right] \quad (3)$$

Adiabatic Process B  $\rightarrow$  C,  $\left\{ \begin{array}{l} T_h V_B^{\gamma-1} = T_c V_c^{\gamma-1} \\ T_h V_A^{\gamma-1} = T_c V_D^{\gamma-1} \end{array} \right.$

Adiabatic Process D  $\rightarrow$  A,  $\left\{ \begin{array}{l} T_h V_B^{\gamma-1} = T_c V_c^{\gamma-1} \\ T_h V_A^{\gamma-1} = T_c V_D^{\gamma-1} \end{array} \right.$

$$\Rightarrow \left( \frac{V_B}{V_A} \right)^{\gamma-1} = \left( \frac{V_c}{V_D} \right)^{\gamma-1}$$

$$\Rightarrow \frac{V_B}{V_A} = \frac{V_c}{V_D} \quad (4)$$

$$(3) \text{ and } (4) \Rightarrow \frac{|Q_c|}{|Q_h|} = \frac{T_c}{T_h}$$

$$\Rightarrow \eta = 1 - \frac{|Q_c|}{|Q_h|} = 1 - \frac{T_c}{T_h}$$

## Carnot Heat Pump COP

$$\begin{aligned} \text{COP} &= \frac{|Q_h|}{W} = \frac{|Q_h|}{|Q_h| - |Q_c|} \\ &= \frac{1}{1 - \frac{|Q_c|}{|Q_h|}} \\ &= \frac{1}{1 - \frac{T_c}{T_h}} \\ &= \frac{T_h}{T_h - T_c} \end{aligned}$$

## Exercise : Efficiency of the Otto Cycle

$$\text{Process } B \rightarrow C, \quad |Q_h| = n C_V (T_C - T_B)$$

$$\text{Process } D \rightarrow A, \quad |Q_c| = n C_V (T_D - T_A)$$

$$\eta = 1 - \frac{|Q_c|}{|Q_h|} = 1 - \frac{T_D - T_A}{T_C - T_B} \quad \textcircled{1}$$

Adiabatic Process :

$$A \rightarrow B : \quad T_A V_A^{\gamma-1} = T_B V_B^{\gamma-1} \quad \textcircled{2}$$

$$C \rightarrow D : \quad T_C V_C^{\gamma-1} = T_D V_D^{\gamma-1} \quad \textcircled{3}$$

$$\text{Also, } \begin{cases} V_A = V_D = V_1 \\ V_B = V_C = V_2 \end{cases}$$

$$\textcircled{2} \Rightarrow T_A = T_B \left( \frac{V_B}{V_A} \right)^{\gamma-1} = T_B \left( \frac{V_2}{V_1} \right)^{\gamma-1} \quad \textcircled{4}$$

$$\textcircled{3} \Rightarrow T_D = T_C \left( \frac{V_C}{V_D} \right)^{\gamma-1} = T_C \left( \frac{V_2}{V_1} \right)^{\gamma-1} \quad \textcircled{5}$$

$$\textcircled{5} - \textcircled{4}, \quad \frac{T_D - T_A}{T_C - T_B} = \left( \frac{V_2}{V_1} \right)^{\gamma-1}$$

$$\Rightarrow \eta = 1 - \left( \frac{V_2}{V_1} \right)^{\gamma-1}$$

$$= 1 - \frac{1}{(V_1/V_2)^{\gamma-1}}$$

## Entropy Change in Free Expansion

$$\Delta S = \int_i^f \frac{dQ_r}{T} = \frac{1}{T} \int_i^f dQ_r$$

Isothermal Process, according to the 1st law,

$$\int_i^f dQ_r = -nRT \ln\left(\frac{V_f}{V_i}\right)$$

$$\Rightarrow \Delta S = \frac{1}{T} \left[ -nRT \ln\left(\frac{V_f}{V_i}\right) \right]$$

$$= nR \ln\left(\frac{V_f}{V_i}\right)$$

## Entropy Generation in a Wall

(i)

$$\dot{S}_{in} - \dot{S}_{out} + \dot{S}_{gen} = \frac{dS_{System}}{dt}$$

Steady State

Rate of net entropy transfer by heat and mass  
 Rate of Entropy Generation

Rate of Change in Entropy

$$\left(\frac{\dot{Q}}{T}\right)_{in} - \left(\frac{\dot{Q}}{T}\right)_{out} + \dot{S}_{gen} = 0$$

$$\frac{1035W}{291} - \frac{1035W}{288} + \dot{S}_{gen} = 0$$

$$\Rightarrow \dot{S}_{gen,wall} = 0.037 \text{ W/K}$$

(ii) Rate of total entropy generation:

$$\frac{1035}{293} - \frac{1035}{283} \rightarrow \dot{S}_{gen} = 0$$

$$\Rightarrow \dot{S}_{gen,total} = 0.125 \text{ W/K}$$